

Deep reinforcement transfer learning of active control for bluff body flows at high Reynolds number

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We demonstrate how to accelerate the computationally taxing process of deep reinforcement learning (DRL) in numerical simulations for active control of bluff body flows at high Reynolds number (*Re*) using transfer learning. We consider the canonical flow past a circular cylinder whose wake is controlled by two small rotating cylinders. We first pre-train the DRL agent using data from inexpensive simulations at low *Re*, and subsequently we train the agent with small data from the simulation at high *Re* (up to $Re = 1.4 \times 10^5$). We apply transfer learning (TL) to three different tasks, the results of which show that TL can greatly reduce the training episodes, while the control method selected by TL is more stable compared with training DRL from scratch. We analyse for the first time the wake flow at $Re = 1.4 \times 10^5$ in detail and discover that the hydrodynamic forces on the two rotating control cylinders are not symmetric.

Key words: drag reduction, turbulence simulation, machine learning

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1. Introduction

Deep reinforcement learning (DRL) has been shown to be an effective way of selecting optimal control of flows in diverse applications (Ren, Hu $&$ Tang [2020\)](#page-15-0), including fish bio-locomotion (Gazzola, Hejazialhosseini & Koumoutsakos [2014;](#page-15-1) Verma, Novati & Koumoutsakos [2018\)](#page-15-2), optimization of aerial/aquatic vehicles' path and motion (Reddy *et al.* [2016;](#page-15-3) Colabrese *et al.* [2017;](#page-15-4) Novati, Mahadevan & Koumoutsakos [2019\)](#page-15-5), active flow control for bluff bodies (Ma *et al.* [2018;](#page-15-6) Bucci *et al.* [2019;](#page-15-7) Rabault *et al.* [2019;](#page-15-8) Ren, Rabault & Tang [2021\)](#page-15-9), shape optimization (Viquerat *et al.* [2021\)](#page-15-10) and learning turbulent wall models (Bae & Koumoutsakos [2022\)](#page-15-11).

The flow past a smooth circular cylinder has been characterized as a 'kaleidoscope' (Morkovin [1964\)](#page-15-12) of interesting fluid mechanics phenomena as the Reynolds number $(Re_D = UD/\nu)$ is increased from 20 to 2×10^5 (*U* is the inflow velocity, *D* is the diameter of the cylinder, ν is the molecular viscosity of the fluid) (Dong *et al.* [2006;](#page-15-13) Cheng *et al.* [2017\)](#page-15-14). The flow develops from two-dimensional steady wake to three-dimensional unsteady vortex shedding, followed by wake transition, shear layer instability and boundary layer transition. Due to the large contrast among flow patterns of different scales, accurate numerical simulations of turbulent flow are usually limited to small and moderate *ReD*. For the same reason, using numerical simulation to study active control of the flow at high *ReD* has rarely been reported.

In our previous study (Fan *et al.* [2020\)](#page-15-15), an efficient DRL algorithm was developed to discover the best strategy to reduce the drag force, by using the DRL to control the rotation of two small cylinders placed symmetrically in the wake of the big cylinder. Specifically, for the flow at $Re_D = 10^4$, it has been demonstrated that DRL can discover the same control strategy as experiments in learning from data generated by high fidelity numerical simulation. Simulated data are noise free, but learning from the simulated data is restricted by the simulation speed. For instance, in Fan *et al.* [\(2020\)](#page-15-15), in the case of $Re_D = 10^4$, it took 3.3 h to generate the simulated data for each episode, and one month to finalize the DRL strategy. In contrast, in companion experimental work it only took a few minutes to learn the same control strategy. As Re_D increases to 1.4×10^5 , one can expect that, even for the simplest task in Fan *et al.* [\(2020\)](#page-15-15), performing DRL from scratch could take months, and hence it might not be practical to apply DRL directly to the more difficult tasks at higher *ReD*.

In this paper, in order to tackle the aforementioned problem, we propose a learning paradigm that first trains the DRL agent using the simulated data at low *Rep* and subsequently transfers the domain knowledge to the learning at higher *Rep*. The rest of the paper is organized as follows: $\S 2$ $\S 2$ gives the details of the simulation model, numerical method and DRL algorithm; § [3](#page-4-0) presents the DRL and transfer learning results for different tasks at three different *Re_D*, namely, *Re_D* = 500, *Re_D* = 10^4 10^4 and *Re_D* = 1.4×10^5 ; § 4 gives the conclusion of the current paper. Finally, [Appendix A](#page-10-0) presents the simulation results of the cases that the control cylinders are rotating at constant speed, while [Appendix B](#page-11-0) presents the validation of the numerical method for simulations at high Reynolds number.

2. Model and methods

In this paper, the bluff body flow control problem has the same geometry as the one in Fan *et al.* [\(2020\)](#page-15-15) but here we focus on demonstrating the feasibility of transferring DRL knowledge from low Re_D to high Re_D , solely in the environment of numerical simulation. As shown in [figure 1,](#page-2-0) the computational model consists of a main cylinder

Figure 1. Sketch of the flow control problem. Here, *U* is the inflow velocity, *D* is the diameter of the main cylinder, $d = \frac{1}{8}D$ is the diameter of the control cylinder, $g = \frac{1}{20}D$, $\theta = 120^\circ$; ω_1 , ω_2 are the angular velocities of control cylinders 1 and 2, respectively. Specifically, $\omega_1 = \epsilon_1 \epsilon_{max}$, $\omega_2 = \epsilon_2 \epsilon_{max}$, where ϵ_1 and $\epsilon_2 \in [0, 1]$ are given by the DRL agent, and ϵ_{max} is a constant.

and two fast rotating smaller control cylinders. This configuration is used to alter the flow pattern around the wake of the big cylinder, with the objective of reducing the effective system drag, or maximizing the system power gain. We note that a similar control strategy has been studied at low Reynolds number and it has been shown that, when the control cylinders are placed at appropriate locations and rotating at a sufficiently fast speed, they are able to change the boundary layers on the main cylinder, i.e. reattach the boundary layer and form a narrower wake, resulting in notable drag reduction.

2.1. *Numerical method*

The numerical simulation of the unsteady incompressible flow past the cylinders of different diameters is achieved by employing the high-order computational fluid dynamics code *Nektar* that employs spectral element discretization on the (*x*–*y*) plane and Fourier expansion along the cylinder axial direction (*z*) (Karniadakis & Sherwin [2005\)](#page-15-16). In particular, we employ entropy viscosity method based large-eddy simulations (LES), which were originally proposed [b](#page-15-18)y Guermond, Pasquetti & Popov $(2011a,b)$ $(2011a,b)$ and later developed further for complex flows by Wang *et al.* [\(2018,](#page-16-0) [2019\)](#page-16-1) and Du *et al.* [\(2023\)](#page-15-19). In the simulations, the computational domain has a size of $[-7.5 D, 20 D] \times [-10 D, 10 D]$ in the *x*, *y* directions, respectively. A uniform inflow boundary condition ($u = U$, $v = 0$, $w = 0$) is prescribed at $x = -7.5 D$, where *u*, *v*, *w* are the three components of the velocity vector *u*. The outflow boundary ($\partial u / \partial n = 0$ and $p = 0$) is imposed on $x = 20 D$, where *p* is the pressure and *n* is the normal vector. A wall boundary condition is applied on the main cylinder surface, while the velocity on the control cylinders are given by the DRL agent during the simulation. Moreover, a periodic boundary condition is assumed on the lateral boundaries ($y = \pm 10 D$). Note that the spanwise length of the computational domain depends on Re_D , namely, it is 6D and 2D in the simulation of $Re_D = 10^4$ and $Re_D = 1.4 \times 10^5$, respectively.

The computational mesh is similar to the one used in Fan *et al.* [\(2020\)](#page-15-15). At $Re_D = 500$, the computational domain is partitioned into 2462 quadrilateral elements, while at $Re_D =$ 10^4 and $Re_D = 1.4 \times 10^5$, it consists of 2790 quadrilateral elements. The elements are clustered around the cylinders in order to resolve the boundary layers. Specifically, on the main cylinder wall-normal directions, the size of the first layer element (Δr) is designed

carefully so that, at $Re = 10^4$, $\Delta r = 4 \times 10^{-3}$; at $Re_D = 1.4 \times 10^5$, $\Delta r = 1.6 \times 10^{-3}$ *D*. On this mesh, with the spectral element mode 4, $y^+ < 1$ can be guaranteed. In the simulation, the time step (Δt) satisfies the Courant condition CFL = $\Delta t |u| / \Delta x \le 0.75$. Note that in all the DRL cases the time duration between two consecutive state queries is fixed at $0.12(D/U)$, e.g. in the case of $Re_D = 1.4 \times 10^5$, $\Delta t = 10^{-4}$, the state data will be collected every 1200 steps. It is worth noting that the reinforcement learning (RL) guided LES starts from fully turbulent flow, which is the result of previous simulation of flow in the same geometric configuration with the small cylinders held still.

2.2. *Deep reinforcement learning and transfer learning*

The DRL identifies the optimal control strategy by maximizing the expected cumulative reward, using the data generated by the simulation. More details of the DRL can be seen in Fan *et al.* [\(2020\)](#page-15-15), but the main principle is summarized as follows:

$$
J(\pi) = \mathbb{E}_{(s_i, a_i) \sim p_{\pi}} \sum_{i=0}^{T} \gamma^i r_i,
$$
\n(2.1)

where J is the so-called cumulative reward, E denotes the calculation of the expected value, $\gamma \in (0, 1]$ is a discount factor and p_{π} denotes the state-action marginals of the trajectory distribution induced by the policy π. Also, $s_i \in S$ is the observed state, $a_i \in A$ is the given actions with respect to the policy $\pi : S \to A$ and r_i is the received reward, at discrete time step *i*. As shown in [\(2.1\)](#page-3-0), the objective of DRL is to discover the policy π_{ϕ} parameterized by ϕ , which maximizes the expected cumulative reward. Specifically, in the current work, the state variable is the concatenation of C_L , C_D , $C_{f,1}$ and $C_{f,2}$, the action is the concatenation of ϵ_1 and ϵ_2 , where C_D and C_L are the drag and lift force coefficient, $C_{f,1}$ and $C_{f,2}$ are the frictional force coefficient on control cylinders 1 and 2, ϵ_1 and ϵ_2 are the action variables for control cylinders 1 and 2, respectively.

In this paper, the twin delayed deep deterministic policy gradient algorithm (TD3) (Fujimoto, Hoof & Meger [2018;](#page-15-20) Fan *et al.* [2020\)](#page-15-15) has been employed. The TD3 consists of two neural networks, one for the actor and the other one for the critic, and both are feedforward neural networks with two hidden layers and 256 neurons. The discount factor $γ$ is set as 0.99. The standard deviation of the policy exploration noise $σ$ is set as 0.005. The Adam optimizer with learning rate 10^{-4} is used, and the batch size is $N = 256$.

Before proceeding to the results, it is worth discussing the difficulties of training the model-free DRL feeding with simulated data. The first difficulty is how to generate more training data as fast as possible. To this end, this paper will propose a multi-client-mono-server DRL paradigm, in which multiple simulations running simultaneously and independently to provide training data to the single DRL server. In each simulation, once a training data are collected, they will be provided to the DRL server. Note that the data exchange between the DRL server and simulation clients is achieved by the XML-RPC protocol.

The second difficulty concerns simulation at high Reynolds number, which makes the multi-client-mono-serve DRL not practical, since a single simulation already requires a significant amount of computing resource. To overcome this difficulty, initially, the DRL agent will be trained using the data collected from the much cheaper simulations at low *ReD*, and then the neural networks will be used in the simulation at high *ReD*, while the network parameters will be re-trained using the new data.

3. Results and discussion

In order to demonstrate the feasibility of using transfer learning (TL) in DRL for active flow control, we consider three different tasks, in which the states, actions and reward functions, corresponding to each task, are given as follows:

- (i) Task 1. Minimizing C_D : two states, C_D and C_L ; two independent actions, ϵ_1 and ϵ_2 ; $r = -\text{sign}(C_D)C_D^2 - 0.1C_L^2$.
- (ii) Task 2. Maximization of the system power gain efficiency: four states, C_D , C_L , $C_{f,1}$ and $C_{f,2}$; one action, $\epsilon_1 = -\epsilon_2$; reward function $r = -\eta$, where

$$
\eta = |C_D| + \frac{\pi d}{D} (|\epsilon_1|^3 C_{f,1} + |\epsilon_2|^3 C_{f,2}) \epsilon_{max}^3.
$$
 (3.1)

(iii) Task 3: maximization of the system power gain efficiency: sates and reward functions are the same as those in task 2; two independent actions, ϵ_1 and ϵ_2 .

3.1. *Learning from scratch at low Re*

We start DRL for task 1 from scratch with $\epsilon_{max} = 5$, in a three-dimensional simulation of flow at $Re_D = 500$. Before presenting the results, we explain why $\epsilon_{max} = 5$ is chosen. As shown in our previous study (Fan *et al.* [2020\)](#page-15-15), C_D decreases with ϵ_{max} , which implies that the generated state variables (C_D, C_L) will be located in a wider range with increasing ϵ_{max} . In order to enhance the generalization of the neural networks, it is beneficial to experience the training data in a wider range before transferring to a more challenging task.

[Figure 2](#page-5-0) shows the evolution of ϵ_1 , ϵ_2 , C_D and C_L , as well as the variation of the vortex shedding pattern due to active control by DRL at $Re_D = 500$. Initially (for 40 episodes approximately), DRL knows very little on how to minimize C_D ; small values of ϵ_1 , ϵ_2 are provided by DRL, thus the control cylinders have a small effect on the flow, hence regular vortex shedding pattern can be observed, as shown by figures $2(A-i)$ $2(A-i)$ and $2(A-i)$. However, around the 40th episode, DRL has figured out the correct rotation direction to reduce C_D , and the wake patterns are now hairpin vortices emanating from the gap between the control and main cylinder. At this stage, $\epsilon_1 \neq \epsilon_2$, and the wake behind the main cylinder is not symmetric, which gives rise to a large C_L , as shown in figures $2(B-i)$ $2(B-i)$ and $2(B-i)$. Around the 110th episode, DRL finally identified the best rotation speed to minimize *CD*, and, correspondingly, the vortex shedding from the main cylinder has been eliminated.

To sum up, when DRL networks are initialized randomly, the DRL agent can gradually manage to learn the optimal policy. The learning process has lasted for more than 100 episodes, and the generated training data, C_D and C_L , are roughly in the ranges $[-0.1, 2.0]$ and $[-1.5, 1.0]$, respectively.

3.2. *Transfer learning from low Re to high Re*

In the previous subsection, we have demonstrated that, for the flow at low Reynolds number, although the training data from the simulation are noise free, it still takes DRL to go through 100 episodes, i.e. 1000 data points, to discover the right policy. Hence, DRL from scratch is impractical to apply to flow at high *Re*, since it will tax the computational resources heavily as it will take much longer to generate the training data. In order to use the data more efficiently and reduce the overall computing time, we employ a TL approach at high *ReD*.

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Figure 2. Task 1: evolution of reinforcement learning and corresponding simulated three-dimensional vortex shedding patterns at $Re_D = 500$, $\epsilon_{max} = 5$. Note that the DRL is starting from scratch, but the simulation is starting from a fully developed flow where the smaller controlling cylinders are stationary.

[Figure 3](#page-6-0) presents the time traces of ϵ_1 , ϵ_2 and C_D of task 1 with $\epsilon_{max} = 3.66$ at $Re_D =$ 10^4 and $Re_D = 1.4 \times 10^5$. In particular, the results of DRL from scratch at $Re_D = 10^4$ are plotted together. Note that, here, both cases of TL were initialized using the same DRL network, which was obtained in task 1 at $Re_D = 500$ shown in figure $2(a)$. We observe that TL spent 60 episodes only to discover the optimal policy, while the DRL from scratch went through over 200 episodes to reach a comparable decision. Moreover, ϵ_1 (pink line, in [figure 3](#page-6-0)*a*) given by the DRL from scratch keeps oscillating even after 500 episodes, but the value given by the TL shows less variation, although the same value of the noise parameter (σ) is used in both cases.

In particular, in the TL at $Re_D = 10^4$, as shown in [figure 3,](#page-6-0) the TL agent manages to find the correct rotation direction in less than 10 episodes, and as it reaches the 20th episode, the TL starts exploring a new policy, during which both ϵ and C_D show notable variations, associated with the so-called 'catastrophic forgetting' (Kirkpatrick *et al.* [2017\)](#page-15-21). After the 55th episode, the TL can make the correct and stable decision.

In task 1, as Re_D is increased from 10⁴ to 1.4 \times 10⁵, the wake behind the main cylinder becomes very complex, although the boundary layer is laminar at both values of *ReD*. The learning process at $Re_D = 1.4 \times 10^5$ is very similar to that of $Re_D = 10^4$. In 25 episodes, the TL is able to identify the correct rotation directions. At the 50th episode, the rotation speed on control cylinder 1 begins to show variation and and it starts a new exploration, due to the catastrophic forgetting. Around the 80th episode, the rotation speed on control cylinder 2 also starts to vary, but the rotation speeds on both control cylinders quickly return to optimal value, i.e. $|\epsilon_{\pm 1}| = 1$.

We would like to emphasize that the high fidelity LES of flow past a cylinder accompanied by two rotating cylinders at $Re_D = 1.4 \times 10^5$ are still a challenge. To the best

Figure 3. Task I: comparison between DRL from scratch and TL at different *Re*. (*a*) Learning from scratch at $Re_D = 10^4$; (*b*) TL $Re_D = 10^4$; (*c*) TL at $Re_D = 1.4 \times 10^5$; (*d*) vortex shedding pattern, the control cylinders are stationary (a,b) , and the pattern at the 125th episode (d) . Note that in $(a-c)$ the pink and black lines are the time traces of actions ($\bar{\epsilon}/\epsilon^{max}$, where $\epsilon_{max} = 3.66$) on control cylinders 1 and 2. Blue line is the time trace of C_D . The agent of the TL was initialized from the saved agent in the previous case at $Re_D = 500$, as shown in [figure 2.](#page-5-0)

of the authors' knowledge, the details of the flow pattern have not been studied before. In figure $3(d)$, the top snapshot exhibits the vortical flow, while the control cylinders remain stationary. The bottom snapshot shows the flow structures at the 125th episode, when the control cylinders are rotating at optimal speed. We see that, as the control cylinder is rotating, the large-scale streamwise braid vortices are mostly replaced by the hairpin vortices emanating from the gap between the main cylinder and control cylinders, and the wake becomes narrower, which is a sign of a smaller C_D .

Next we turn our attention to task 2, where the objective is to maximize the system power gain efficiency, η , under the condition that $\epsilon_2 = -\epsilon_1$. Note that the objective of the current task 2 is similar to the task 2 in our previous study (Fan *et al.* [2020\)](#page-15-15), except that, here, we have used the instantaneous C_f obtained from the simulation, while C_f is a pre-defined constant in Fan *et al.* [\(2020\)](#page-15-15). Nonetheless, our previous study has revealed that it is much more difficult for DRL to identify the optimal control strategy in task 2 than that in task 1. In task 2, we have started from the DRL from scratch using two-dimensional (2-D) simulation at $Re_D = 500$. In particular, since the 2-D simulation is relatively inexpensive and fast, we have run 16 simulations concurrently to provide the training data to a single DRL. For the TL at $Re_D = 10^4$ and $Re_D = 1.4 \times 10^5$, the simulation is very expensive, thus only a single simulation was performed. It is worth noting that η at low $\mathbb{R}e_D$ is very different from that at high *ReD*, as shown in [Appendix A,](#page-10-0) thus it is expected that TL in task 2 and task 3 will be more challenging.

[Figure 4\(](#page-8-0)*a*) shows the learning process of DRL from scratch for task 2 at $Re_D = 500$. In particular, the learning process using a single 2-D simulation $(a-c)$ and 16 2-D simulations $(d-f)$ are plotted together. In figure $4(a)$, DRL manages to find the correct rotation direction after 75 episodes, but it barely makes the optimal decision, although it has been trained over 300 episodes. On the other hand, in the case of 16 simulations running concurrently, as shown in [figure 4\(](#page-8-0)*d*), DRL can identify the correct rotation direction in less than 10 episodes, and is able to make the optimal decision before the 75th episode.

Subsequently, the DRL networks trained in figure $4(a,d)$ are both applied to the TL at $Re_D = 10^4$, and the results are shown in figure $4(b,e)$. We observe from figure $4(b)$ that, although the DRL of figure (*a*) has identified the correct policy, it gives a very chaotic ϵ_1 in the first 60 episodes in TL. On the other hand, as shown in figure $4(e)$, the TL is able to give the correct rotation direction in less than 10 episodes, and after 30 episodes it has managed to reach the optimal decision. When applied to the simulation flow at $Re = 1.4 \times 10^5$, the TL manages to give stable action values after the 30th episode, but in a short period just after the 40th episode and until the end of the simulation, an unstable action is given, as shown by the variation of ϵ_1 and η in [figure 4\(](#page-8-0)*c*). To explain the variation of the decision given by the agent, the policy at 5th, 15th, 25th and 65th episodes are visualized in figure $4(f)$. Specifically, the black boxes $[-0.25, 0.25] \times [0.3, 0.8]$, which correspond to the concentrated intervals of C_L and C_D , respectively, are highlighted. We observe that the policy in the highlighted regions has not reached the best strategy yet, although the TL has been trained for 70 episodes. Next, we consider the hardest problem of this paper, task 3, with the same objective as that of task 1, but ϵ_1 and ϵ_2 are independent. Again, we start this task from scratch in a 2-D simulation at $Re_D = 500$, as shown in figure $5(a,b)$. Note that, here, 16 2-D simulations were used to provide training data. As shown in figure $5(a,b)$, the RL agent was able to give correct rotation directions for both controlling cylinders after 50 episodes. Between the 50th and 110th episodes, ϵ_2 gradually approaches -1 , while ϵ_1 oscillates around 0.7, the combination of which gives rise to $\eta \approx 1.5$. After the 110th episode, the DRL suddenly changes its actions, in a short period of exploration: ϵ_1 changes to 1, ϵ_2 is -0.5 and the value of η goes down to 1.37. Meanwhile, the DRL does not stop exploration on ϵ_1 , as shown by the variation in the black line, starting around the 200th episode. Around the 280th episode, the DRL is able to reach a near optimal decision, as demonstrated by the fact that η is close to 1.325, which is the optimal value obtained from task 2.

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Figure 4. Task 2: TL from 2-D low Rep to 3-D high $Rep. (a,d)$ Learning from scratch, $Rep = 500$: left – single client; right – multi-client. Panels (b,e) show TL, $Re_D = 10^4$: (*a*) and (*d*) are initialized by the corresponding DRL agents shown in panels (*a*,*d*), respectively. Panels (*c*,*f*) show TL, $Re_D = 1.4 \times 10^5$, initialized from the agent corresponding to (*e*). Panel (*f*) shows visualization of the policy at the 5th, 15th, 25th and 65th episodes. Note in all the simulations here $\epsilon_{max} = 3.66$.

The TL results of task 3 at $Re_D = 10^4$ are plotted in [figure 5\(](#page-9-1)*b,d*). We observe that TL is able to reach the correct decision on rotation directions in less than 10 episodes. As more training data are obtained, ϵ_2 keeps the maximum value 1, and ϵ_1 is gradually increased to around $\epsilon_1 = -0.8$, which gives rise to $\eta \approx 0.84$, which is slightly greater than the optimal value $\eta = 0.78$, obtained in task 2. The policy contours at the 5th, 50th, 110th, 220th and 285th episodes for $Re_D = 500$ and the 5th, 30th, 80th, 100th and 140th episodes for $Re_D =$ 10^4 are plotted in [figures 5\(](#page-9-1)*c*) and [5\(](#page-9-1)*d*), respectively. During the learning process, at $Re_D =$ 500, *C_D* and *C_L* are mostly in the range [0.1, 0.6] and [−1.25, −0.75], and at $Re_D = 10^4$, these two coefficients are mostly in the range [0.3, 0.8] and [−0.5, 0.0], respectively. The corresponding *CD*, *CL* ranges are highlighted by black boxes in the two figures. We observe

Figure 5. Task 3: ϵ_1 and ϵ_2 are independent. (*a*,*c*) The DRL from scratch using 16 2-D simulations at Re_D = 500; (*b*,*d*) TL at $Re_D = 10^4$. Panels (*c*,*d*) where the top row refers to ϵ_1 and the bottom row refers to ϵ_2 , shows the policy at the episodes indicated by the black circles in the corresponding figure of upper panel, respectively.

that, at both $\mathcal{R}e_D$, the policies for ϵ_1 and ϵ_2 both gradually approach the value that results in optimal η , which clearly shows the exploration and exploitation stages at work.

4. Summary

We have implemented a DRL in the numerical simulation of bluff body flow active control at high Reynolds number ($Re_D = 1.4 \times 10^5$). We demonstrated that, by training the DRL using the simulation data at low Re_D , and then applying TL at high Re_D , the overall learning process can be accelerated substantially. In addition, the study shows that the TL can result in a more stable decision, which is potentially beneficial to the flow control. Moreover, we proposed a multi-client-single-server DRL paradigm that is able to generate training data much faster to quickly discover an optimal policy. While here we focus on a specific external flow, we believe that similar conclusions are valid for wall-bounded flows and different control strategies.

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Figure 6. Time series of C_D and C_L on the main and control cylinders: (a,c) $Re_D = 10^4$; (b,d) $Re_D = 1.4 \times$ 10⁵. Note that the control cylinders are rotating at constant speed at $\epsilon_1 = -1$, $\epsilon_2 = 1$, $\epsilon_{max=3.66}$. The red lines are the hydrodynamic force coefficients on the main cylinder, while the blue and pink ones are on control cylinders 1 and 2, respectively.

Author contributions. M.T., G.E.K. and D.F. conceived of the presented idea. M.T. and G.E.K. supervised the findings of this work. Z.W. wrote the code and performed the numerical simulations. X.J. encouraged Z.W. to investigate and provided the funding. All authors contributed to the final version of the manuscript.

Appendix A. Simulation results of control cylinders rotating at constant speed

In this section, the simulation results for the cases where both control cylinders are rotating at constant speed $|\Omega| = \epsilon_{max} = 3.66$. [Figure 6\(](#page-10-1)*a–d*) shows C_D and C_L on the three cylinders at $Re_D = 10^4$ and $Re_D = 1.4 \times 10^5$, respectively. We observe that, with rotating control cylinders, the C_D on the main cylinder at both Re_D is reduced significantly. With ϵ_{max} = 3.66, the control cylinders are not able to cancel the vortex shedding on the main cylinder, thus C_L on the main cylinder at both Re_D exhibits the frequency of vortex shedding. In particular, at $Re_D = 1.4 \times 10^5$, the magnitudes of C_L on control cylinders 1 and 2 show notable discrepancy from each other, which leads to symmetry breaking on the average at this Re_D . In addition, [figure 7](#page-11-1) plots C_f on the control cylinders at both Re_D . As shown in the figure, at both Re_D , on average, the magnitudes of C_f on the control cylinders are different.

In order to validate the optimal control strategy given by DRL in task 2 and task 3, additional simulations with $\epsilon_2 = 0$, $\epsilon_2 = 0.6$, $\epsilon_2 = 0.75$, $\epsilon_2 = 0.9$ and $\epsilon_2 = 1.0$ for $Re_D = 500$, and $\epsilon_2 = 0$, $\epsilon_2 = 0.85$, $\epsilon_2 = 0.9$ and $\epsilon_2 = 1.0$ for $Re_D = 10^4$ are performed. [Figure 8](#page-11-2) plots the power gain coefficient η as a function of ϵ_2 . Note that the value of η at

Figure 7. Time series of frictional coefficient (C_f) on the control cylinders at $Re_D = 10^4$ and $Re_D = 1.4 \times 10^5$, $\epsilon_1 = 1$, $\epsilon_2 = 1$, $\epsilon_{max=3.66}$.

Figure 8. Power gain coefficient (*η*) varies with ϵ : blue, $Re_D = 500$; red, $Re_D = 10^4$. Note that, at $Re_D = 10^4$, the value of η has been enlarged by three times. Here, $\epsilon_{max} = 3.66$.

 $Re = 10⁴$ has been enlarged by three times in order to show the variation more clearly. We observe that the minimum of η for $Re_D = 500$ is around $\epsilon_2 = 0.75$, and it is $\epsilon_2 = 0.9$ for $Re_D = 10^4$.

Appendix B. Validation of simulation of cylinder flow at high *Re*_D

In order to validate the numerical method (spectral element plus entropy–viscosity), LES of flow past a cylinder at $Re_D = 1.4 \times 10^5$ have been performed. The computational domain has a size of $[-12 D, 16 D] \times [-10 D, 10 D]$ in the streamwise (*x*) and cross-flow (*y*) directions, respectively. The domain is partitioned into 2044 quadrilateral elements. The size of elements around the cylinder in the radial direction is 0.0016 *D* in order to resolve the boundary layer. The domain size in the spanwise (*z*) direction is 3*D*. Uniform inflow velocity is prescribed at the inflow boundary, a homogeneous Neumann boundary

Table 1. Sensitivity study of the simulation result to the mesh resolution: flow past a single circular cylinder at $Re_D = 1.4 \times 10^5$. Here, $\overline{C_D}$ is the mean drag coefficient, $\overline{C_L}$ is the root-mean-square value of the lift coefficient, *St* is the Strouhal number, L_r is the length of the recirculation bubble and ϕ_s is the separation angle. The (Breuer [2000\)](#page-15-22) LES is case D2. LES¹ and LES² correspond to the mesh resolutions 1 and 2, respectively.

Figure 9. Local pressure and skin friction coefficient at $Re_D = 1.4 \times 10^5$; comparison with the literature. Note that 'Present LES' is from the simulation using mesh LES², 'Cantwell & Coles experiment' refers to the experiment by Cantwell & Coles [\(1983\)](#page-15-24), 'Achenbach experiment' refers to the experiment by Achenbach [\(1968\)](#page-15-25).

condition for velocity and zero pressure is imposed at the outflow boundary, and wall boundary condition is imposed at the cylinder surface and a periodic boundary condition is assumed at all other boundaries. In particular, two simulations of different resolution are performed: LES¹, third-order spectral element, 120 Fourier planes, $\Delta t = 1.5 \times 10^{-4}$; LES², fourth-order spectral element and 160 Fourier planes, $\Delta t = 1.0 \times 10^{-4}$.

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Figure 10. Validation of the LES: (*a*) mean centreline velocity \bar{u}/U_{∞} ; (*b*) mean velocity \bar{u}/U_{∞} at $x/D = 1$; (*c*) mean turbulent shear stress $\overline{u'v'}/U_{\infty}^2$ at $x/D = 1$. Note that 'Present LES' is from the simulation using mesh LES², 'Cantwell & Coles experiment' refers to the experiment by Cantwell & Coles [\(1983\)](#page-15-24).

[Table 1](#page-12-0) presents the statistical values of $\overline{C_D}$, $\overline{C_L}$, *St*, L_r and ϕ_s from the simulation at $Re_D = 1.4 \times 10^5$. We observe that the simulation results agree with the values obtained from literature very well, and the results are not sensitive to the mesh resolution. A further validation with the experimental measurement is shown in [figure 9,](#page-12-1) which plots the mean local C_p and C_f . Again, the simulation results agree with experimental measurement very well. [Figure 10](#page-13-0) exhibits the comparison between present LES and the experimental measurement by Cantwell & Coles (1983) . It could be observed that the LES results of the mean streamwise velocity \bar{u}/U_{∞} along horizontal line $y = 0$ and vertical line $x/D = 1$ are in good agreement with the experiments. [Figure 11](#page-14-0) shows the streamlines and contours of the mean velocity and Reynolds stress in the near wake. The results shown in the figures agree with figures 5 and 6 of Braza, Perrin & Hoarau [\(2006\)](#page-15-23) well.

Figure 11. Validation of the LES: mean velocity and Reynolds stress field. Note that the reference results are shown in the figures agree with figures 5 and 6 of Braza *et al.* [\(2006\)](#page-15-23). The present simulation results are from the simulation using mesh resolution LES². (*a*) Streamlines; (*b*) \bar{u} ; (*c*) \bar{v} ; (*d*) $\overline{u}\overline{u}$; (*e*) $\overline{v}\overline{v}$; (*f*) $\overline{u}\overline{v}$.

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